

Lecture 6-7-8 Flexural Members

- ✓ **-**I- Laterally Restrained Beams
- ✓ II- Laterally Unrestrained Beams

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Introduction: Beams, Response to loads

لمَـنارة A beam is a structural member which is subjected to transverse loads, and accordingly must be designed to withstand predominantly shear and moment, Generally, it will be bent $w = 84$ kN/m about its major axis..

Slender structural elements loaded in a stiff plane tend to fail by buckling in a more flexible plane (out-of-plane buckling)

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Introduction: Unrestrained Beams

- **this lecture covers the design of unrestrained beams that are prone to lateral torsional buckling.**
- **Beams without continuous lateral restraint are prone to buckling about their major axis, this mode of buckling is called lateral torsional buckling (LTB).**

Lateral torsional buckling can be discounted when:

- **The section is bent about its minor axis**
- **Full lateral restraint is provided**
- **Closely spaced bracing is provided making the slenderness of the weakaxis low**
- **The compressive flange is restrained again torsion**
- **The section has a high torsional and lateral bending stiffness**

Introduction: Unrestrained Beams Behaviour

Beam is Perfectly elastic, initially straight, loaded by equal and opposite end moments about its major axis.

- ✓ **Beam is Unrestricted along its length.**
- ✓ **End Supports**
	- ✓**Twisting and lateral deflection prevented.**
	- ✓**Free to rotate both in the plane of the web and on plan.**

Introduction: Unrestrained Beams

Beam is Perfectly elastic, initially straight, loaded by equal and opposite end moments about its major axis.

Three components of displacement are observed i.e

- **Vertical (y)**
- • **Horizontal (x)**
- • **and torsional () displacement**

Introduction: Unrestrained Beams-Elastic Critical Moment

 M_{ν}

 \mathcal{C}

Elastic critical moment

Consider the following assumptions:

- **Perfect beam, without any type of imperfections (geometrical or material);**
- **Doubly symmetric cross section;**
- **Material with linear elastic behavior;**
- **Small displacements (cos(** ϕ **)=1 ;** $\sin(\phi) = \phi$)

The critical value of the moment about the major axis My , denoted as M^E cr (critical moment of the "standard case") resulting in lateral torsional buckling is obtained:

L

$$
M_{cr}^{E} = \frac{\pi}{L} \sqrt{G I_T E I_z \left(1 + \frac{\pi^2 E I_W}{L^2 G I_T}\right)},
$$

Introduction: Unrestrained Beams-Elastic Critical Moment Elastic critical moment

It can be observed that the critical moment of a member under bending depends on several factors, such as:

- **loading (shape of the bending moment diagram);**
- **support conditions;**
- **length of the member between laterally braced cross sections;**
- **lateral bending stiffness; torsion stiffness; warping stiffness.**

Besides these factors, the point of application of the loading also has a .directinfluence on the elastic critical moment of a beam

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Real Steel Beams

- **In reality beams are not free from imperfection, not purely elastic, not always simply supported, not always loaded with only a constant flexure and are not of a doubly symmetric sections, consequently, subject to different bending moment diagrams.**
- **The derivation of an exact expression for the critical moment for each case of real beams is not practical, as this implies the computation of differential equations of some complexity.**
- **Therefore, in practical applications approximate formulae are used, which are applicable to a wide set of situations.**

Real Steel Beams

As an alternative to some of the expressions, the elastic critical moment can be estimated using expression below proposed by Clark and Hill (1960) and Galea (1981) . It is applicable to members subject to bending about the strong axis, with cross sections mono-symmetric about the weak z axis, for several support conditions and types of loading.

$$
M_{cr} = C_1 \frac{\pi^2 E I_z}{(k_z L)^2} \left[\left[\left(\frac{k_z}{k_w} \right)^2 \frac{I_w}{I_z} + \frac{(k_z L)^2 G I_T}{\pi^2 E I_z} + (C_2 z_g - C_3 z_j)^2 \right]^{-0.5} \right],
$$

\n
$$
- (C_2 z_g - C_3 z_j)
$$

\n
$$
\vdots
$$
 is the distance between points of lateral restraint (L_{cr})
\nis the Young's Modulus = 210000 N/mm²

- is the shear modulus = 80770 N/mm²
- is the second moment of area about the weak axis
- is the torsion constant
- is the warping constant
- is an effective length factor related to rotations at the end section about the k, weak axis z (can be conservatively taken as 1.0)
	- is an effective length factor related to warping restriction in the same cross sections (can be conservatively taken as 1.0)

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Real Steel Beams

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$$

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Real Steel Beams

As an alternative to some of the expressions, the elastic critical moment can be estimated using expression below proposed by Clark and Hill (1960) and Galea (1981) . It is applicable to members subject to bending about the strong axis, with cross sections mono-symmetric about the weak z axis, for several support conditions and types of loading.

figure.

<https://manara.edu.sy/> Steel Structures2 Prof.Dr. Nael M. Hasan **zj = 0 0for beams with doubly symmetric cross section (such as Ior H cross sections with equal flanges)**

C1, C2, andC3 are coefficients depending on the shape of the bending moment diagram and on support conditions,

C1, C2, andC3 are coefficients depending on the shape of the bending moment diagram and on support conditions,

• In beams subject to end moments, by definition $C_2 Z_g = 0$.

• $\psi_f = \frac{I_{fc} - I_{fi}}{I_{fc} + I_{fi}}$, where I_{fc} and I_{fi} are the second moments of area of the

compression and tension flanges respectively, relative to the weak axis of the section (z $axis);$

• C_1 must be divided by 1.05 when $\frac{\pi}{k_w L} \sqrt{\frac{E I_w}{G I_T}} \le 1.0$, but $C_1 \ge 1.0$.

C1, C2, andC3 are coefficients depending on the shape of the bending moment diagram and on support conditions,

In case of mono-symmetric I or H cross sections, the tables can be used if the following condition is verified

$$
-0.9 \leq \psi \leq 0.9
$$

Resistance of Real Steel Beams

Real beams differ from an ideal beams in much the same way as do real compression members. • Thus any small imperfections such as

Lateral deflection and twist

- initial crookedness, twist,eccentricity of load, or horizontal load components cause thebeam to behave as if it had an equivalent initial crookedness and twist, as shown by curve A
- Imperfections such as residual stresses or variations in materialproperties cause the beam to behave as shown by curve B.
- The behavior of real beams having both types of imperfection isindicated by curve C.
- Curve C shows a transition from the elastic behaviour of a beam with curvature and twist to the inelastic postbuckling behaviour ofa beam with

residual stresses.

The influence of Slenderness

- **Considering the analogy between Ncr and Mcr, the lateraltorsional behavior of beams in bending is similar to a compressed column. Therefore:**
- **The resistance of short/stocky members depends on the value ofthe cross section bending resistance (plastic or elastic bending moment resistance, depending of its cross section class).**
- **The resistance of slender members depends on the value of the critical moment (Mcr), associated with lateraltorsional buckling.**
- **The resistance of members with intermediate slenderness depends on the interaction between plasticity and instability**

The influence of Slenderness

Non-dimensional plot permits results from different test series to be compared.

- **Stocky beams** (λ _{LT}<0.4) unaffected by lateral torsional buckling
- **Slender beams** (λιτ>1.2) resistance close to elastic critical moment Mcr.
- **Intermediate slenderness** adversely affected by inelasticity and geometric imperfections.
- **EC3 uses a reduction factor χ_{LT} on plastic resistance moment to cover the whole slenderness range.**.

The influence of Slenderness

Summary of factors to consider influence of Slenderness

Warping: is the distortion of the elements of a steel section out of the plane perpendicular to the axis of the member under twisting/torsion.

Restraining this effects will have a favorable impact in avoiding lateral torsional buckling

End Constraints: Restraints have a major influence on the occurrence of instability and can be utilized to enhance the load carrying capacity of the beam whenever instability is likely to occur.

The stiffness in the minor axis Vs stiffness in the major axis: Section with relatively equal stiffness about both axis are almost never likelyto experience LTB. **Bracing:** Lateral bracing of beams is the common measure to overcome the occurrence of LTB

Point of Load application: In relation to the shear center of the section the point of load application may have a favorable/stabilizing or unfavorable/destabilizing effect

Lateral-Torsional Buckling Resistance

The verification of resistance to lateral-torsional buckling of a prismatic member consists of the verification of the following condition (clause6.3.2.1(1)):

$$
\frac{M_{Ed}}{M_{b, Rd}} \leq 1.0 \; ,
$$

Mb,Rd is the design buckling resistance, given by (clause6.3.2.1(3))

where : $W_y = W_{p1,y}$ for class 1 and 2 cross sections; $W_y = W_{\text{el},y}$ for class 3 cross sections; $W_v = W_{eff,v}$ for class 4 cross sections; X_{1T} is the reduction factor for lateral-torsional buckling.

In EC3-1-1 two methods for the calculation of the reduction coefficient χ_{LT} in prismatic members are proposed:

A General Method that can be applied to any type of cross section (more conservative)

Alternative Method that can be applied to rolled cross sections or equivalent welded sections.

A General Method-Any section

$$
\chi_{LT} = \frac{1}{\phi_{LT} + (\phi_{LT}^2 - \overline{\lambda}_{LT}^2)^{0.5}}, \quad \text{but } \chi_{LT} \le 1.0,
$$

$$
\phi_{LT} = 0.5 \left[1 + \alpha_{LT} \left(\overline{\lambda}_{LT} - 0.2 \right) + \overline{\lambda}_{LT}^2 \right],
$$

$$
\overline{\lambda}_{LT} = \left[W_y f_y / M_{cr} \right]^{0.5}
$$

 α L_T is the imperfection factor, which depends on the buckling curve 0.21, 0.34, 0.49 and 0.76 for curves a, b, c and d

M_{cr} is the elastic critical moment.

The buckling curves to be adopted depend on the geometry of the cross section of the member

Alternative Method-Rolled or equivalent welded sections

Students are highly advised to read more on this topic. The discussion of this method presented in "*Design of Steel Structures Eurocode 3, 2010, by da Silva L.S.* " is recommended as a starting literature.

Deflection Resistance

- **Deflections of flexural members must be limited to avoid damage to finishes, ceilings and partitions, and should be calculated under SLS loads.**
- **EC3 states that limits for vertical deflections should be specified for each project and agreed with the client. The UK National Annex to EC3 suggests:**

Vertical deflections [BS EN 1993-1-1:2005, 7.2.1(1)B] NA.2.23

The following table gives suggested limits for calculated vertical deflections of certain members under the characteristic load combination due to variable loads and should not include permanent loads. Circumstances may arise where greater or lesser values would be more appropriate. Other members may also need deflection limits.

On low pitch and flat roofs the possibility of ponding should be investigated.

Standard rules for maximum deflection:

BEAM BENDING

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Deflection Resistance Summary

- 1. Define Service loads (Actions)
- 2. Define Section and beam prosperities
- 3. Draw the bending moment diagram
- 4. Determine Maximum deflection of beam
- 5. Determine Deflection limits
- 6. Compare Maximum deflection of beam with Deflection limits

Conditions for ignoring the lateral-torsional buckling verification

The verification of lateral-torsional buckling for a member in bending may be ignored if at least one of the following conditions is verified:

$$
\overline{\lambda}_{LT} \leq \overline{\lambda}_{LT,0} \text{ or } M_{Ed} / M_{cr} \leq \overline{\lambda}_{LT,0}^2
$$

Where; $|||||||| || \overline{\lambda}_{LT,0} = 0,4$ (maximum value)

Improving the lateral torsional buckling resistance

In practical situations, for given geometrical conditions, support conditions and assumed loading, the lateral-torsional buckling behaviour of a member can be improved in two ways:

- by increasing the lateral bending and/or torsional stiffness, by increasing the section or changing from IPE profiles to HEA or HEB or to closed hollow sections (square, rectangular orcircular);
- by laterally bracing along the member the compressed part of the section (the compressed flange in the case of I or H sections). This is more economical, although sometimes it is not feasible.

Bending Moment Resistance Summary:

- 1.Draw the bending moment diagram to obtain the value of the maximum bending moment, M_{Ed} .
- 2. Determine fy and calculate the class of the section. Once you know the class of the section then you will know which value of the section modulus you will need to use in the equation for $M_{b,Rd}$.
- 3. Work out the effective length, L_{cr} .
- 4. Work out the value of M_{cr} , the critical moment.
- 5.Work out the lateral torsional slenderness ratio using either the
	- general case or alternative expression.

6. Work out Φ_{LT} using either the general case or alternative expression. 7. Work out X_{1T} using either the general case or alternative expression. 8. Calculate the design buckling resistance $M_{c,Rd}$. 9.Carry out the buckling resistance **Mc,Rd > MEd**.

Example4.4.

Consider the beam, supported by web cleats and loaded by two concentrated loads, P=70.0kN (design loads). Design the beam usinga HEA profile, inS235 steel (E=210GPa and G=81GPa), according to EC3-1-1. Consider free rotation at the supports with respect to the y-axis and the z-axis. Also assume free warping at the supports but consider that the web cleats donot allow rotation around the axis of the beam (x axis). Assume:

a) Unbraced beam;

b) Beam is braced at points of application of the concentrated loads.

The cross section class of a HEA 240 is obtained as follows Web in bending, $\frac{c}{t} = \frac{164}{7.5} = 21.9 < 72 \varepsilon = 72 \times 1 = 72.0$ Flange in compression, The HEA 240 is class 1. confirming the use of $W_{\text{ol,v}}$ $\frac{c}{t} = \frac{240/2 - 7.5/2 - 21}{12} = 7.9 < 9 \varepsilon = 9 \times 1 = 9$ S 235 for t≤16mm **Material Properties: HEA 240** • $f_v = 235 \text{ MPa}$ ► W_{pl, y}=744.6cm³ ► 1_T=41.55cm⁴ ► f_u = 235 MPa
► 1_y=7763cm⁴ ► 1_w=328.5 X 10³ cm⁶ ► E = 210 Gpa $1 = 2769cm⁴$ \triangleright G=81 GPa Step3: Check for Lateral-torsional buckling without intermediate bracing [a]. $\frac{M_{Ed}}{M} \leq 1.0$, $M_{b, Rd}$ Step3.1: Compute the buckling resistance $M_{b, Rd} = \chi_{LT} W_{v} f_{v} / \gamma_{M1}$, $W_v = W_{pl,v}$ for class 1= 744.6cm³

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 z_i = 0 for beams with doubly symmetric cross section (such as I or H cross sections with equal flanges)

 $C_1 = 1.04$, $C_2 = 0.42$ and $C_3 = 0.562$

Loading and Diagram of $k C_1$ \mathbb{C}^{I} $C₂$ support conditions moments 0.42 0.562 1.0 1.04 0.95 0.31 0.539 0.5 1 $M_{cr} = 231.5 \text{ kNm} \implies \overline{\lambda}_{LT} = 0.87$. 2 Since $\alpha_{LT} = 0.21$ (H rolled section, with $h/b \le 2$) 3 $\phi_{LT} = 0.95 \implies \chi_{LT} = 0.75$ 4 Compute the buckling resistance $M_{b, Rd} = \chi_{LT} W_v f_v / \gamma_{M1}$, $M_{b,kd} = 0.75 \times 744.6 \times 10^{-6} \times \frac{235 \times 10^3}{1.0} = 131.2$ kNm > $M_{Ed} = 105.0$ kNm **O.K.**

solution :b

Step4: Check for Lateral-torsional buckling with intermediate restraints [b].

Step4.1: Compute the buckling resistance

If the beam is laterally braced at the points of application of the loads, the lateral-torsional buckling behavior is improved.

Deflection Verification: SLS unfactored imposed actions.

Unfactored variable loads are shown below

W = 30 kN, a=1.5 m, L=6m, E= 210000 N/mm², I=7763 10⁴ mm⁴

$$
\delta = \frac{wal^2}{12EI} = \frac{30000x1500x6000^2}{12x210000x77630000} = 8.28 \text{ mm}
$$

Vertical deflection limit:

$$
\frac{L}{360} = \frac{6000}{360} = 16.7 \text{ mm} \rightarrow 16.67 \text{ mm} > 8.28 \text{ mm} \text{ O.K.}
$$

$$
\chi_{LT} = \frac{1}{\phi_{LT} + (\phi_{LT}^2 - \overline{\lambda}_{LT}^2)^{0.5}}, \quad \text{but } \chi_{LT} \le 1.0,
$$
\n
$$
\phi_{LT} = 0.5 \left[1 + \alpha_{LT} \left(\overline{\lambda}_{LT} - 0.2 \right) + \overline{\lambda}_{LT}^2 \right];
$$
\n
$$
\overline{\lambda}_{LT} = \left[W_y f_y / M_{cr} \right]^{0.5}
$$
\n
$$
M_{cr} = C_1 \frac{\pi^2 E I_z}{(k_z L)^2} \left\{ \left[\left(\frac{k_z}{k_w} \right)^2 \frac{I_w}{I_z} + \frac{(k_z L)^2 G I_r}{\pi^2 E I_z} + (C_2 z_g - C_3 z_j)^2 \right]^{0.5} \right\},
$$
\n
$$
= \left[C_2 z_g - C_3 z_j \right]
$$

 $k_z = k_w = 1.0$, as the standard case support

 $=$ 0, The elastic critical moment of the beam is not z_{n} aggravated by the fact that the loads are applied at the upper flange, because these are applied at sections that are laterally restrained.

 $W_v = W_{\text{pl},v}$ for class 1= 568.5cm³

 z_i = 0 for beams with doubly symmetric cross section (such as I or H cross sections with equal flanges) $C_1 = 1.00$, $C_2 =$ not important as $Z_0 = 0$ and $C_3 = 1.0$ Diagram of $k_{\rm r}$ | C_1 Loading and \mathcal{C}_3 support conditions moments $\psi_i \leq 0$ $\psi_f > 0$ $\Psi = +1$ 1.00 1.0 1.000 $0.5 \, | \, 1.05 \, |$ 1.019 $\overline{\lambda}_{LT} = 0.49$. M_{cr} = 551.3 kNm As $\alpha_{LT} = 0.21$ (rolled H section, with $h/b \le 2$), $\phi_{LT} = 0.65 \Rightarrow \chi_{LT} = 0.93.$ $M_{b, Rd} = 0.93 \times 568.5 \times 10^{-6} \times \frac{235 \times 10^3}{1.0} = 124.2$ kNm > $M_{Ed} = 105.0$ kNm **O.K.**

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Deflection Verification: SLS unfactored imposed actions.

Unfactored variable loads are shown below

W = 35 kN, a=1.5 m, L=6m, E= 210000 N/mm², I=54100000 10⁴ mm⁴

 $\delta = \frac{wal^2}{12E}$ 12 = $35000x1500x6000^2$ $\frac{330001130016000}{12x210000x54100000} = 13.86$ mm

Vertical deflection limit:

$$
\frac{L}{360} = \frac{6000}{360} = 16.7 \text{ mm} \qquad \rightarrow \rightarrow 16.67 \text{ mm} > 13.86 \text{ mm} \text{ O.K.}
$$

Summary

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Summary

